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## 2 Multi-criteria optimal design of parallel manipulators based 3 on interval analysis

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### 8 Abstract

9 In optimal design problems we have to determine a set of design parameters such that a given mechanism  
10 satisfies a list of requirements. In practice however these requirements may be classified as either *compulsory*  
11 or *relaxable*. For classical optimal design methodologies, it is very difficult to find the solutions that satisfy  
12 compulsory requirements simultaneously and make the best compromise between these two kinds of  
13 requirements. So in this paper we propose and illustrate on an example of parallel robots an approach  
14 based on interval analysis that allows to determine almost *all* possible mechanism geometries such that  
15 all compulsory requirements will be satisfied simultaneously. As using interval analysis all possible solu-  
16 tions will be obtained as a set of regions in the parameter spaces, the best design compromise for the relax-  
17 able requirements will be determined by sampling the solution regions.

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19 *Keywords:* Parallel manipulators; Interval analysis; Optimization

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### 21 1. Introduction

22 Compared with serial manipulators, parallel manipulators have many advantages such as high-  
23 er rigidity, better positioning accuracy, high speed and high load capacity. But one of their draw-

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2

*F. Hao, J.-P. Merlet / Mechanism and Machine Theory xxx (2004) xxx–xxx*

24 backs is that their performances depend heavily on their geometry. So optimal design has become  
25 a key issue for their development and many researchers have recently paid attention to this prob-  
26 lem [1–6].

27 A classical way to solve optimal design problems is to define a real-valued function  $C$  as a  
28 weighted sum of requirement indices  $I_i$ , which are functions of the design parameter set  $\mathcal{P}$ .

$$C = \sum_i w_i I_i(\mathcal{P}) \quad (1)$$

31 where  $w_i$  are weights. Numerical procedures are then used to find a set  $\mathcal{P}_m$  which minimizes  $C$ ,  
32 usually starting with an initial guess  $\mathcal{P}_0$ .  $\mathcal{P}_m$  will be considered as the optimal design solution.

33 But this method has many drawbacks. First it is assumed that the requirement indices can be  
34 defined and that they can be calculated efficiently as the numerical optimization procedure re-  
35 quires a large number of evaluation of these indices. But these assumptions are difficult to satisfy  
36 in practice especially for parallel robots: for example what could be the definition of an index indi-  
37 cating that a cube with a given volume must be included in the workspace of the mechanism? As  
38 for the performance evaluation it is now well known that *exact* performance evaluation is very  
39 difficult to obtain because of complexity reasons or numerical round-off errors in the calculation.

40 A second problem of cost-function approaches is that the numerical optimization procedure  
41 may converge toward a local minimum, which leads to a solution that may be quite far away from  
42 the optimal design solution.

43 In cost-function approaches it is difficult to determine the weights, because the weights not only  
44 indicate the priority of the requirements but also tackle with the unit problems of the performance  
45 indices. For example for a 3-DOF translational robot if the used performance indices are the  
46 workspace volume and positioning accuracy, then for a fair comparison between both criteria  
47 a weight with ratio  $10^3$  must be chosen. Furthermore a small change in the weights may lead  
48 to very different optimal design solutions. But until now there are not intuitive rules for determin-  
49 ing their values.

50 The cost-function approaches may also lead to inconclusive results. It was exemplified by  
51 Stoughton [7] who was going to design a special kind of Gough platforms with improved dexterity  
52 and a reasonable workspace volume. He found that these criteria were varying in opposite direc-  
53 tions: the dexterity was decreasing when the workspace volume was increasing. Hence the prob-  
54 lem of optimal design has become the problem of determining an acceptable compromise between  
55 the two requirements. In most literature authors have been aware of this problem and consider  
56 only one performance criterion (see for example [8–18]). But for practical applications it is quite  
57 seldom that there is only one criterion to be considered (see for example [19–21]).

58 Another major drawback of cost-function approaches is that it provides only one optimal de-  
59 sign solution. In our opinion an optimal design methodology should provide a set of solutions for  
60 the following reasons:

- A designer may not have all the necessary information to make the final design choice (for example he may obtain an arbitrary length for a linear actuator while the end-user will finally decide to use a commercially available actuator with only finite possibilities for the lengths).

- In general there is not a unique solution to a design problem as compromises have to be made. Providing various solutions allows the end user to choose the best design compromise for the problem at hand.

67

68 Finally introducing manufacturing errors in the cost-function is difficult. This function may be  
 69 very sensitive to design errors, so the final instance of the real mechanism has performances that  
 70 are quite different from the theoretical optimal design.

71 In this paper we will consider a difficult problem of optimal design of a 6-DOF parallel manip-  
 72 ulator with multi-criteria requirements. The contribution of this paper is summarized as follows:

- A new optimal design methodology based on interval analysis is proposed which allows to determine almost all the possible geometries satisfying two compulsory requirements (on the workspace and accuracy).
- Two alternative approaches are introduced to determine the best design compromise for the relaxable requirements.
- A simplified algorithm which is reasonable and acceptable in practice is proposed to speed-up computation.

## 81 2. Parallel manipulator kinematics

82 The mechanical architecture of the considered robot is presented in Fig. 1. It is known as “ac-  
 83 tive wrist” and has been patented by INRIA in 1991 [22]. A mobile platform is connected to six  
 84 fixed length legs through ball-and-socket joints. An universal joint is located at the other extrem-  
 85 ity of the leg and the location of the joint center can be changed via the motion of a linear actuator  
 86 connected to the base (in the considered design the motion axis of the joint is vertical to the base,

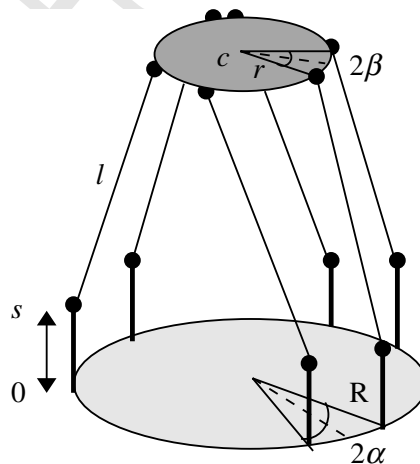


Fig. 1. The parallel manipulator with fixed actuators.

4

*F. Hao, J.-P. Merlet / Mechanism and Machine Theory xxx (2004) xxx-xxx*

87 but this direction may be arbitrary). Controlling the locations of the U-joints allows to control the  
88 pose of the mobile platform.

89 *2.1. Design parameters and workspace definition*

90 For simplicity, it is assumed that the attachment points of all the joint centers lie on a circle on  
91 the base and the mobile platform. The joint centers are symmetrical with respect to three lines  
92 located 120° apart. So the geometry of such parallel manipulators is defined by six design  
93 parameters:

- $R, r$ : the radii of the circles on which lie the attachment points of the legs on the base and platform;
- $\alpha, \beta$ : the half angles between two adjacent attachment points on the base and platform;
- $s$ : the stroke of the actuator;
- $l$ : the length of the leg (supposed to be identical for all legs).

99  
100 The desired workspace of the robot is denoted as  $\mathcal{W}$  and is decomposed into a desired *orien-*  
101 *tation workspace* and a *translation workspace*. The desired orientation workspace is defined by  
102 three ranges of the pitch, roll, yaw angles  $\phi_1, \phi_2, \phi_3$ . Similarly the translation workspace is defined  
103 by three ranges of the  $x, y, z$  coordinates of the reference point  $C$  on the platform.

104 For identical heights of the actuated joints the  $x, y$  coordinates will be 0 while the  $z$  coordinates  
105 will depend on the design parameters. Hence the desired range of the  $z$  coordinates is defined as a  
106 relative motion  $z_r$  around the *nominal height*  $z_n$ , which is the  $z$  coordinates of the reference point  $C$   
107 when all the actuators are at their mid stroke  $\rho_m$ . We have:

$$z_n = \rho_m + \sqrt{l^2 - R^2 - r^2 + 2Rr \cos(\gamma)} \quad (2)$$

110 where

$$\rho_m = \frac{s}{2} \quad (3)$$

$$\gamma = \frac{\pi}{3} - \alpha - \beta \quad (4)$$

115 so the  $z$  coordinates in the reference frame is

$$z = z_n + z_r \quad (5)$$

118 *2.2. Robot kinematics*

119 In this paper it is assumed that the workspace of the parallel manipulator is limited only by the  
120 motion ranges of the actuators. Note however that other constraints (such as limited motion of  
121 the passive joints) can be considered easily.

122 From Fig. 2, we have

$$B_i - A_i = [d_x \quad d_y \quad d_z - \rho_i] \quad (6)$$

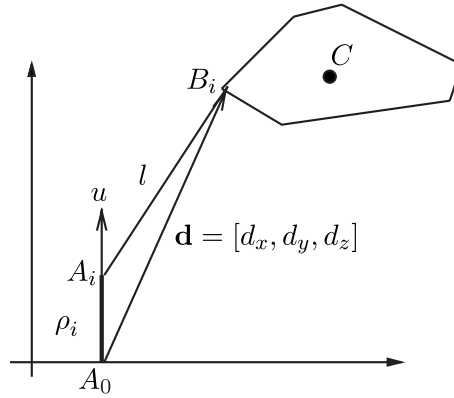


Fig. 2. The leg of the parallel manipulator.

125 where  $B_i$ ,  $A_i$  are the extremity coordinates of the legs.  $d_x$ ,  $d_y$ ,  $d_z$  are known for a given pose of the  
126 platform and  $\rho_i$  is the length of the actuated link. As the norm of  $B_i - A_i$  must be equal to  $l$ , we get

$$\rho = d_z \pm \sqrt{l^2 - d_x^2 - d_y^2} \quad (i = 1 \dots 6) \quad (7)$$

129 Between the two possible solutions we select

$$\rho = d_z - \sqrt{l^2 - d_x^2 - d_y^2} \quad (i = 1 \dots 6) \quad (8)$$

132 For a given pose if

$$\rho_{\min} = 0 \leq \rho_i \leq s = \rho_{\max} \quad (9)$$

135 is verified, then this pose can be reached by the manipulator.

### 136 2.3. Error analysis and singularity

137 The positioning accuracy of the platform is influenced by a set of parameter errors  $\Delta\Theta$  such as  
138 measurement errors of the actuated joints, location errors of the attachment points, etc. As these  
139 errors are usually small a linear error model is used:

$$\Delta q = J(p, q)\Delta\Theta \quad (10)$$

142 where  $q$  are the poses of the platform and  $p$  are the geometry parameters. For a full error model  
143  $\Delta\Theta$  will be a large vector, but it has been recognized that the measurement errors of the actuator  
144 motions induce the largest errors on the positioning of the platform [23]. Hence the influence of  
145 the other parameters is neglected, so

$$\Delta q = J(p, q)\Delta\rho \quad (11)$$

149 where  $J(p, q)$  is  $6 \times 6$  Jacobian matrix whose inverse  $J^{-1}$  is defined by

$$\left[ \begin{array}{cc} \mathbf{A}_i \mathbf{B}_i & \mathbf{A}_i \mathbf{B}_i \times \mathbf{B}_i \mathbf{C} \\ \mathbf{u} \cdot \mathbf{A}_i \mathbf{B}_i & \mathbf{u} \cdot \mathbf{A}_i \mathbf{B}_i \end{array} \right] \quad (12)$$

6

*F. Hao, J.-P. Merlet / Mechanism and Machine Theory xxx (2004) xxx-xxx*

152 The inverse Jacobian matrix is important for singularity analysis. But in this paper we will not  
153 consider the singularity problem because we have already designed an algorithm that allows to  
154 check whether there is singularity in a given workspace for a family of robots whose geometry  
155 parameters are denned by a set of ranges [24].

156 It is well known that determining a closed-form of the Jacobian matrix  $J(p, q)$  is very difficult.  
157 Hence a difficult problem of error analysis is how to express the positioning errors analytically as  
158 a function of the sensor errors.

### 159 3. Interval analysis

#### 160 3.1. Interval arithmetics

161 Interval arithmetics is a simple method that can provide lower and upper bounds for a function  
162 with interval unknowns. One of its important advantages is that it allows computer round-off er-  
163 rors to be taken into account. The *interval evaluation* of a function determines an interval that  
164 guarantees the inclusion of the exact lower and upper bounds of this function. The simplest inter-  
165 val evaluation method is the *natural evaluation* in which each mathematical operator  $\diamond$  of the  
166 function is replaced by an interval equivalent  $\diamond'$  returning an interval  $[\underline{\diamond'}, \overline{\diamond'}]$  such that for all  
167  $x$ : in a range  $X$ ,  $\underline{\diamond'}(X) \leq \diamond(x) \leq \overline{\diamond'}(X)$ .

168 Consider for example the function

$$f(x) = x^2 + x \tag{13}$$

172 for  $x$  in  $[-1, 1]$ . The interval equivalent of the square function is denned by

$$[a, b]^2 = \begin{cases} [0, \text{Max}(a^2, b^2)] & \text{if } 0 \in [a, b] \\ [\text{Min}(a^2, b^2), \text{Max}(a^2, b^2)] & \text{otherwise} \end{cases} \tag{14}$$

175 Hence

$$f([-1, 1]) = [0, 1] + [-1, 1] = [-1, 2] \tag{15}$$

178 Note that the interval evaluation of a function depends heavily on its analytical form. For exam-  
179 ple Eq. (13) is rewritten as

$$f(x) = \left(x + \frac{1}{2}\right)^2 - \frac{1}{4} \tag{16}$$

182 Using this form, we have

$$f([-1, 1]) = \left[-\frac{1}{2}, \frac{3}{2}\right]^2 - \frac{1}{4} = \left[-\frac{1}{4}, 2\right] \tag{17}$$

185 3.2. Notations for interval analysis

186 The lower and upper bounds of an interval  $X$  will be denoted by  $\underline{X}, \bar{X}$  and the width of this  
187 interval is  $w(X) = \bar{X} - \underline{X}$ . The midpoint of an interval  $X$  is defined as

$$\text{mid}(X) = \frac{\bar{X} + \underline{X}}{2} \quad (18)$$

190 An  $n$ -dimensional interval set is called a *Box*:

$$X = \{[\underline{X}_1, \bar{X}_1], \dots, [\underline{X}_n, \bar{X}_n]\} \quad (19)$$

193 The width  $w$  of an  $n$ -dimensional interval set  $X$  is the maximal width of its interval components.

194 Bisection is one of the most basic operation of interval analysis. For an  $n$ -dimensional interval  
195 set  $X$  the results of a bisection along the variable  $x_i$  are two new interval sets  $L(X), R(X)$  defined by

$$L(X) \triangleq \{[\underline{x}_1, \bar{x}_1], \dots, [\underline{x}_i, (\underline{x}_i + \bar{x}_i)/2], \dots, [\underline{x}_n, \bar{x}_n]\} \quad (20)$$

$$R(X) \triangleq \{[\underline{x}_1, \bar{x}_1], \dots, [(\underline{x}_i + \bar{x}_i)/2, \bar{x}_i], \dots, [\underline{x}_n, \bar{x}_n]\} \quad (21)$$

200 4. Optimal design for the workspace requirement

201 In order to use interval analysis we must be able to define an initial range for every design  
202 parameter (which is a reasonable assumption in most cases). So the *allowed parameter box*  
203 (APB) is defined as an  $n$ -dimensional box that contains all the allowable values of the design  
204 parameters.

205 The *feasible parameter boxes* (FPBs) are defined as boxes such that any point belonging to a  
206 FPB defines a geometry of the mechanism that satisfies one of the compulsory requirements. In  
207 our approach FPBs will be determined by using interval analysis and their union will be an  
208 approximation of the region that represents all the mechanisms satisfying one of the compulsory  
209 requirements.

210 The *valid parameter boxes* (VPBs) are the intersections of all FPBs of different compulsory  
211 requirements. Points in VPBs define mechanism geometries that satisfy all compulsory require-  
212 ments simultaneously.

213 4.1. Determination of the allowed parameter box

214 In most cases it is possible to obtain initial bounds for the design parameters:

- 0 is an evident lower bound for  $r, R$ , but consideration on the interference between the passive joints will lead to a better lower bound. The overall size of the manipulator provides an upper bound. Note that for symmetry reasons an additional constraint is  $r \leq R$ .
- $\alpha, \beta$  have at least 0 as the lower bound but consideration on the interference between the passive joints will also lead to a better lower bound. The upper bound which is  $\alpha, \beta \leq \frac{\pi}{3}$  is obtained by considering the symmetry of the attachment point locations.

8

*F. Hao, J.-P. Merlet / Mechanism and Machine Theory xxx (2004) xxx–xxx*

- $l$  has a lower bound which is  $R - r$ , The upper bound is obtained by considering the maximal, size of the robot (note that the accuracy requirement will enable to eliminate too large values of  $l$  quickly as the dexterity will be poor in that case).
- $s$  has a lower bound which is the required maximum travel in  $z$  direction, the upper bound is determined either by constraints on the overall size or by constraints on commercially available components that will be used in the mechanism.

228 The desired workspace  $\mathcal{W}$  will be provided by the user. Without lack of generality we assume  
229 that  $\mathcal{W}$  is denned by specified ranges for each pose parameter (but other workspace shape can be  
230 used as well).

#### 231 4.2. Algorithm principle

232 In order to satisfy the workspace requirement it is necessary to verify that the length of the  
233 actuated link  $\rho_i$  satisfies

$$\rho_{\min} = 0 \leq \rho_i \leq \underline{s} = \rho_{\max} \quad (22)$$

236 for any pose in the workspace. A first algorithm  $F_w(P, Q)$  based on interval analysis will take as  
237 inputs a design parameter box  $P$  and a pose parameter box  $Q$  included in the desired workspace  
238  $\mathcal{W}$ . In  $F_w(P, Q)$  we compute the interval evaluation  $[\underline{\rho}_i, \overline{\rho}_i]$  for the six actuated links and  $F_w(P, Q)$   
239 will return:

- 240 •  $-1$  if  $\underline{\rho} > \rho_{\max}$  or  $\overline{\rho} < \rho_{\min}$  for at least one leg and at least one pose in  $Q$ ;
- 241 •  $0$  if  $\underline{\rho} < \rho_{\min}$  or  $\overline{\rho} > \rho_{\max}$  for at least one leg and one pose in  $Q$ ;
- 242 •  $1$  if  $\underline{\rho} \geq \rho_{\min}$  and  $\overline{\rho} \leq \rho_{\max}$  for all legs and all poses in  $Q$ .

243 If  $F_w(P, Q)$  returns 1, then any robot whose design parameters lie within  $P$  can reach all the poses  
244 included in  $Q$ . If  $F_w(P, Q)$  returns  $-1$ , then the design parameters included in  $P$  define the parallel  
245 manipulators whose geometries do not allow to reach some poses in the desired workspace  $\mathcal{W}$ . If  
246  $F_w(P, Q)$  returns 0 then we cannot decide whether the design parameters in  $P$  (or in some parts of  
247  $P$ ) define the right robot geometries or not, as the overestimation of interval arithmetics may be  
248 the reason why  $\underline{\rho} < \rho_{\min}$  or  $\overline{\rho} > \rho_{\max}$ .

249 Our main algorithm will determine a set of boxes  $P_i$  such that  $F_w(P_i, Q_j)$  will return 1 for all  
250 elements  $Q_j$  of a set whose union is the desired workspace  $\mathcal{W}$ . During the calculation boxes  $P_i$  with  
251 width lower than a given threshold  $\epsilon$  will be called *neglected boxes* and will not be considered  
252 (although they may be stored in some particular structures): the main motivation for neglecting  
253 boxes is to take into account manufacturing errors by choosing  $\epsilon$  equal to twice the manufacturing  
254 errors. Indeed because of manufacturing tolerances, only the FPB whose width is at least twice the  
255 manufacturing errors can guarantee that the geometry parameters of the real mechanism lie in the  
256 FPB when choosing the center of the FPB as the manufacturing parameters.

257 The algorithm that determines the FPBs for the workspace requirement is based on  $F_w(Q, P)$   
258 and is similar to the algorithm presented in [25]. It uses a list  $\mathcal{L} = \{P_i\}$  of  $n$  potential FPBs ini-  
259 tialized with the APBs, a list  $\mathcal{S} = \{Q_j\}$  of  $m$  boxes of the pose parameters and thresholds  $\epsilon, \omega$  of



260 the width  $w$  of the boxes  $P_i, Q_j$ . During each bisection of  $P_i$  two elements are added to  $\mathcal{L}$  and are  
 261 placed at the end of the list. Similarly each bisection of  $Q_j$  adds two elements to  $\mathcal{L}$ . The algorithm  
 262 proceeds as follows:

- 263 1. loop 1  
 264 (a) if  $i > n$ , then EXIT  
 265 (b) if  $F_w(P_i, \mathcal{W}) = -1$ , then  $i = i + 1$ , go to 1(a)  
 266 (c) if  $F_w(P_i, \mathcal{W}) = 1$ , then store  $P_i$  as a FPB,  $i = i + 1$ , go to 1(a)  
 267 (d) if  $w(P_i) < \epsilon$ , then store  $P_i$  as a neglected box,  $i = i + 1$ , go to 1(a). Otherwise go to loop 2  
 269 2. loop 2, with  $\mathcal{S} = \{Q_1 = \mathcal{W}\}, j = m = 1$   
 270 (a) if  $j > m$ , then store  $P_i$  as a FPB,  $i = i + 1$ , go to 1(a)  
 271 (b) if  $w(Q_j) < \varpi$ , then bisect  $P_i$ ,  $n = n + 2$ ,  $i = i + 1$  and go to 1(a)  
 272 (c) if  $F_w(P_i, Q_j) = -1$ , then  $P_i$  cannot be a FPB,  $i = i + 1$ , go to 1(a)  
 273 (d) if  $F_w(P_i, Q_j) = 1$ , then  $j = j + 1$ , go to 2(a)  
 274 (e) bisect  $Q_j$ ,  $j = j + 1$ ,  $m = m + 2$ , go to 2(a)  
 275 (f) end of loop 2  
 277 3. end of loop 1

278  
 279 The above algorithm guarantees that all the FPBs of the desired workspace with width larger  
 280 than  $\epsilon$  can be determined. Note that this algorithm is an incremental algorithm. Indeed we usually  
 281 start the calculation with a large value of  $\epsilon$  and then refine the calculation with a lower value of  $\epsilon$   
 282 only using the boxes that have been neglected during the previous run as the initial data of the list  
 283  $\mathcal{L}$ , thereby reducing a large number of calculation.

284 In this algorithm other constraints on the workspaces can also be taken into account, for exam-  
 285 ple motion ranges of passive joints. Indeed such constraints can be defined by an inequality con-  
 286 straint  $\mathcal{G}(P, Q) \leq 0$ . In that case  $F_w(P, Q)$  will return:

- 287 • 1 if  $\underline{\rho} \geq \rho_{\min}$  and  $\bar{\rho} \leq \rho_{\max}$  and  $\mathcal{G}(P, Q) \leq 0$  for all legs and all poses in  $Q$ ;  
 288 •  $-1$  if  $\underline{\rho} > \rho_{\max}$  or  $\bar{\rho} < \rho_{\min}$  or  $\mathcal{G}(P, Q) > 0$  for at least one leg and at least one pose in  $Q$ ;  
 289 • 0 otherwise.

290  
 291 Using the same principle  $F_w(P, Q)$  can be extended to deal with arbitrary specification of the  
 292 desired workspace  $\mathcal{W}$  as soon as a test  $\mathcal{T}(Q)$  has been defined that returns 1 if the set of pose  
 293 parameters  $Q$  belong to  $\mathcal{W}$ , otherwise it returns 0. In that case  $F_w(P, Q)$  will return 1 or  $-1$  only  
 294 when  $\mathcal{T}(Q) = 1$ .

## 295 5. Optimal design for the accuracy requirement

296 A classical requirement for accuracy is that the positioning errors of the platform should be less  
 297 than the fixed threshold  $\Delta X$ , being given the range  $\Delta \rho^m = [\underline{\Delta \rho^m}, \overline{\Delta \rho^m}]$  of the measurement errors  
 298 on the locations of the actuated joints.

299 Eq. (11) could be used but as mentioned before the evaluation of Jacobian matrix  $J(x)$  is very  
300 difficult especially for 6-DOF parallel robots. On the other hand, the inverse Jacobian matrix  
301  $J^{-1}(x)$  can be calculated easily in a closed form. So from Eq. (11), we have

$$\Delta\rho = J^{-1}(P, Q)\Delta Q \quad (23)$$

304 Assume that  $P, Q$  are defined as ranges and  $T_{ij}$  is denoted as the absolute value of the interval  
305 evaluation of the element of  $J^{-1}(P, Q)$  at the  $i$ th row and  $j$ th column. Then an interval  $U_i$  is de-  
306 fined as

$$U_i = \sum_{k=1}^{k=6} T_{ik} \overline{\Delta X}_k \quad (24)$$

309 Clearly  $\overline{U}_i$  is an upper bound of the maximal allowable value for  $\Delta\rho_i$  such that the positioning  
310 errors of the platform do not exceed  $\Delta X$ . Similarly  $\underline{U}_i$  is a lower bound of this value.

311 Similar to  $F_w(P, Q)$ , an algorithm  $F_a(P, Q)$  is designed that takes as inputs a design parameter  
312 box  $P$  and a pose parameter box  $Q$  included in the desired workspace  $\mathcal{W}$  and will return:

- $-1$  if there exists  $i$  such that  $\overline{U}_i < \Delta\rho^m$ . In that case for any robot geometry included in  $P$  and for any pose within  $Q$ , the worst allowable accuracy necessary to obtain the required positioning errors  $\Delta X$  is lower than  $\Delta\rho^m$ .
- if  $\underline{U}_i \leq \Delta\rho^m$  for all  $i$ . In that case for any parameters included in  $P$  and  $Q$  the best allowable accuracy necessary to obtain the required positioning errors  $\Delta X$  is greater than  $\Delta\rho^m$ . Consequently the positioning errors at any pose in  $Q$  induced by sensor errors (bounded by  $\Delta\rho^m$ ) will always be lower than  $\Delta X$ .
- 0 otherwise.

321  
322 Using  $F_a(P, Q)$  instead of  $F_w(P, Q)$  in the algorithm presented in Section 4.2 we obtain an algo-  
323 rithm that allows to determine an approximation of all the robot geometries such that the accu-  
324 racy requirement will be satisfied over the workspace  $\mathcal{W}$ .

## 325 6. Calculation of the valid parameter boxes

326 The algorithms allow us to get the FPBs of the workspace and accuracy requirements as a list of  
327 design parameter boxes. Hence computing the intersection of the two sets of FPBs is straightfor-  
328 ward and so is obtaining the VPBs. Two alternative strategies may be used to speed-up the  
329 computation:

1. Use the FPBs of the workspace requirement as the initial list  $\mathcal{L}$  to verify the accuracy require-  
ment. In that case the FBPBs of the accuracy requirement will be the final VPBs.
2. Modify the algorithm principle to perform  $F_w(P, Q)$  and  $F_a(P, Q)$  simultaneously so that both  
the workspace and accuracy requirements will be checked, allowing the direct calculation of  
the VPBs.

336 **7. Determining optimal parameter values**

337 *7.1. Design for relaxable requirements*

338 Using the previous algorithms all the possible design solutions (VPBs) that fulfill all the com-  
 339 pulsory requirements are obtained. But each design solution in VPBs may present very different  
 340 performances apart from the compulsory requirements (Fig. 3), hence relaxable requirements such  
 341 as inertia, cost, bandwidth, stiffness, etc. may be considered. Two alternative approaches will be  
 342 used for these types of requirements:

1. *Sampling of the VPBs.* In this approach each box in the VPBs is sampled, providing a list of possible design solutions. The relaxable requirements are evaluated for each design solution, which provides a set of design solutions with various compromises for the relaxable requirements.
2. *Relaxed VPB.* VPBs can be computed also for the relaxed version of the relaxable requirements. Then the intersection of all VPBs will provide all the design solutions that satisfy both the compulsory requirements and the relaxed version of the relaxable requirements.

350

351 *7.2. Computing the intersection of sorted lists*

352 The motivation in both cases is to provide a list of design parameters presenting different com-  
 353 promises among relaxable requirements. For each relaxable requirement or other additional  
 354 requirements we sort the design parameters  $p_i$  with descending order according to their perform-  
 355 ance indices and then compute the intersection of the top part of each list (Fig. 4), if the intersec-

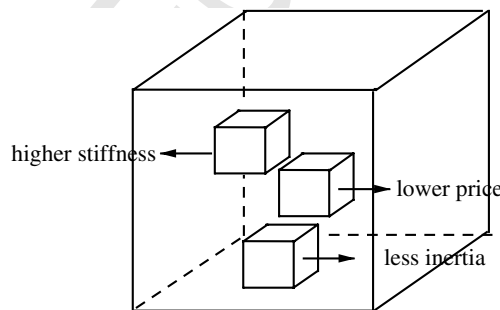


Fig. 3. Performances of each solution in VPBs.

Lower inertia	$p_8$	$p_2$	$p_3$	$p_4$	$p_1$	...
Higher stiffness	$p_4$	$p_3$	$p_6$	$p_7$	$p_8$	....
Lower price	$p_2$	$p_5$	$p_8$	$p_4$	$p_6$	...

Fig. 4. Sorted design parameter lists.

356 tion is not empty, then it gives the end-user several optimal parameter values (for instance  $p_4, p_8$ )  
357 with concrete indices to make the final decision, even considering some additional requirements  
358 that have not been specified during the design stage. It is clearly an advantage of our approach.

## 359 8. Algorithm in practice

360 The algorithms presented in the previous sections can provide all the design solutions that sat-  
361 isfy all the compulsory requirements simultaneously. But checking all the requirements over the  
362 full workspace is very computer intensive as the number of the unknowns (the design parameters  
363 and the pose parameters) may be large. Note that the algorithm presented in Section 4.2 also can  
364 be used to verify the compulsory requirements for a fixed value of the design parameters, with a  
365 greatly reduced computation time. Hence a simplified algorithm is used in practice to speed-up the  
366 computation:

1. Compute the VPBs for a relaxed version of the compulsory requirements at some specified poses.
2. Sample the obtained VPBs to get a set of potential design solutions.
3. Use the algorithm to verify each potential design solution over the whole workspace of the robot.

372  
373 When performing step 3, instead of checking the requirements over the full desired workspace  
374 we propose to check them only in a limited set  $\mathcal{R}$  of *check segments* connecting specific poses,  
375 called *check points*, belonging to  $\mathcal{W}$ . Hence loop 2 of the algorithm is replaced by a simpler loop  
376 performed along the check segments in  $\mathcal{R}$ . Each pose  $M$  on a segment connecting check points  
377  $M_1, M_2$  is described by

$$\mathbf{OM} = \mathbf{OM}_1 + \lambda \mathbf{M}_1 \mathbf{M}_2 \quad (25)$$

381 where  $\lambda$  is an interval parameter in the range  $[0, 1]$ . Hence the number of the unknowns in loop 2 is  
382 reduced from 6 (the pose parameters) to 1 ( $\lambda$ ). The obtained VPBs are called the *relaxed VPBs* and  
383 are constituted of:

- the true VPBs;
- boxes defining geometries that satisfy the compulsory requirements at all poses in  $\mathcal{R}$  but may not satisfy them over the full  $\mathcal{W}$ .

387

## 388 9. Application examples

389 This section presents a numerical example of optimal design based on our approach. It has been  
390 implemented using the high level interval analysis package ALIAS<sup>1</sup> which relies on the C++ inter-  
391 val arithmetics package BIAS/Profil.

<sup>1</sup> [www.inria-sop.fr/coprin/logiciel/ALIAS/ALIAS.html](http://www.inria-sop.fr/coprin/logiciel/ALIAS/ALIAS.html).

Table 1  
Specification for the design parameters

	$R$ (mm)	$r$ (mm)	$\alpha$ (deg)	$\beta$ (deg)	$l$ (mm)	$s$ (mm)
Lower bound	500	130	10	10	500	1400
Upper bound	550	180	30	30	1200	2200
$\epsilon$	10	10	2.8	2.8	100	100

392 The ranges and accuracy  $\epsilon$  of the design parameters are presented in Table 1. The desired work-  
 393 space is defined as a hyper-cube whose center is the nominal position. The vertices of the desired  
 394 6-D workspace are chosen as the check points, whose coordinates are defined by  
 395  $\{x_i, y_j, z_{rk}, \phi_1, \phi_2, \phi_3\}$  ( $i, j, k = 1, 2$ ), where  $x \in \{-100, 100\}$ ,  $y \in \{-100, 100\}$ ,  $z_r \in \{-500, 500\}$ ,  
 396  $\phi_1 = \phi_2 = -p/9$ ,  $\phi_3 = -p/6$ . The check segments connecting the pair of check points ( $i, j$ )  
 397 where  $\{i = 1 \dots 8, j = i + 1 \dots 8\}$  are constructed based on equation (25). A very preliminary  
 398 sequential implementation for the workspace requirement has been tested. The computation time  
 399 is less than 48 hours on a PC (2.00 GHz) under Linux. All together, 13455 boxes have been tested  
 400 and 8781 valid parameter boxes have been obtained with a total volume of  $1.13958e+08$ , the ne-  
 401 glected volume consisting of  $\epsilon$  boxes is  $1.40415e+07$ . We are quite confident that by using other  
 402 powerful methods of interval analysis [27,28] the computation time can be reduced drastically.

403 A preliminary accuracy analysis has been performed by sampling the FPBs of the workspace  
 404 and computing the worst case of positioning errors at the check points. It is noticed that the posi-  
 405 tioning accuracy seems to be very sensitive to the design parameters  $\alpha, \beta$  and less sensitive to the  
 406 other design parameters: the positioning errors increase with the parameters  $\alpha, \beta$  (this observation  
 407 is coherent with the works presented in [26,7]). As mentioned earlier requirements of workspaces  
 408 and accuracy seem to be antagonistic. Therefore the result of a cost-function approach satisfying  
 409 both requirements only reflects the relative weights that are used in the cost-function while our  
 410 approach allows to obtain the design solutions that satisfy both a minimal workspace and optimal  
 411 accuracy or the opposite.

## 412 10. Conclusion

413 A new methodology is proposed in this paper for the optimal design of parallel manipulators  
 414 with multi-criteria requirements. The main differences with other classical approaches are that this  
 415 methodology allows to obtain *all* the possible design solutions that satisfy a set of compulsory  
 416 requirements (taking into account manufacturing errors) and make the best compromise for  
 417 the relaxable requirements.

418 The prospective works are:

- 419 1. Improving the current implementation to reduce the computation time. For example note that  
 verifying the violation or satisfaction of the requirement for a given  $Q_k$  is independent from  
 the other  $Q_l$  of the list, so a distributed implementation can be used (and is available within  
 ALIAS). A distributed implementation may reduce the computation time by a greater factor  
 than the number of slave computers.

14

*F. Hao, J.-P. Merlet / Mechanism and Machine Theory xxx (2004) xxx–xxx*

2. Automatizing the treatment of the workspace requirement so that other mechanical architectures can be treated as well. Indeed for any mechanical architecture we have a set  $P$  of design parameters, a set  $\mathcal{S}$  of pose parameters and the robot workspace may be defined by the set of poses  $Q$  that satisfy some constraint relations  $\mathcal{F}(P, Q) \leq 0$ . The only differences between two mechanical architectures are the parameters in  $P$ ,  $Q$  and the relations in  $\mathcal{F}$ . But the principle of our methodology is still valid for any architectures as soon as  $P, Q, \mathcal{F}$  have been defined. Hence a symbolic pre-processing may be used to generate automatically this architecture-dependent module whose results will be taken as arguments for an optimal design kernel, thereby allowing to deal with any mechanical architectures with a minimum effort.
3. Extending the compulsory requirements to process other classical performance indices such as stiffness, joint forces/torques, joint velocities, etc.

435

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