Curved mesh generation via linear elasticity. Applications to fluid dynamics problems.

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Motivation

- Many researches to design method for fluid problems with accuracy higher than 2nd order
- True accuracy of those method requires that the mesh boundary are represented with at least the same accuracy



Subsonic flow around 2 cylinders: spurious entropy production

 \Rightarrow Curved meshes fitting well the boundaries: Bézier/NURBS approximation with triangular patches

Guideline

Bézier and NURBS approximation

Curvilinear mesh generation

Mesh checking Computing curved mesh via linear elasticity analogy

Numerical simulations

Isogeometric analysis for unstructured meshes

Bézier basis functions of order k over a simplex

- Lets λ_i the barycentric coordinates,
- α a multi-index of length k,
- the Bézier polynomial is:

$$B_{\alpha}(M) := c(\alpha) \prod_{i=1}^{d+1} \lambda_i(M)^{\alpha_i}$$

 \mathcal{NURBS} basis function of order k

$$N_{\alpha} = \frac{\omega_{\alpha} B_{\alpha}}{\sum\limits_{\alpha', |\alpha'| = k} \omega_{\alpha'} B_{\alpha'}^k}.$$

 $\left\{ \begin{array}{ll} N_{\alpha}(M) & \geq 0 \text{ for any } \alpha, |\alpha| = k \text{ and } M \in K \\ \\ \sum_{\alpha, |\alpha| = k} & N_{\alpha}(M) = 1 \end{array} \right.$

Isogeometric analysis for unstructured meshes Lets $\mathcal{P} = \{P_{\alpha} \in \mathbb{R}^{p}, \alpha \in \mathcal{I}\}$, a family of control points, we approximate a function ψ by

$$\psi(M) \approx \sum_{\alpha, \alpha \in \mathcal{I}} P_{\alpha} \varphi_{\alpha}^{n}(M).$$

- The basis functions can be defined on triangles/tetra and quadrangulars,
- they allow to define conics.

Example of elements



Property of curved element

Convexity of control polygon

A \mathcal{NURBS} curve is contained in the convexe hull of its control polygon



Recalls

- extrapolation of control point
- the end points are always interpolated

Curved mesh generation

Problematic

• How to generate \mathcal{NURBS} meshes ?

Our approach

• Starting from a classical mesh of the domain (piecewise linear), we modify it to generate *NURBS* mesh.

2 main steps

- Is the geometrical information: control point, weight..
 ⇐⇒ curve the boundary mesh
- Check validity of volumic elements and if needed curve the interior mesh

From a piecewise linear mesh to a \mathcal{NURBS} mesh Straight mesh and curved boundary.



How to curve a piecewise linear mesh ?

Given

a simplicial mesh + the definition of boundary geometry

• Is the curved mesh conformal?



How to curve a piecewise linear mesh ?

Given

a simplicial mesh + the definition of boundary geometry

- Is the curved mesh conformal?
- How to curve the volumic mesh with keep as much as possible the structure of the mesh ?



How to detect automatically invalid element?

A valid straight element



but.... a non-valid curved element



How to detect automatically invalid element?

Convex hull property



How to detect automatically invalid element?



 $\alpha_2 \leq 0 \Rightarrow \text{invalid element}$

How to curve a piecewise linear mesh ?

Given

a simplicial mesh + the definition of boundary geometry

• Is the curved mesh conformal? Compute signed area for each triangle which have a boundary edge:

- if all area ≤ 0 , the mesh is ok
- else
- e How to curve the volumic mesh with keep as much as possible the structure of the mesh ?

How to curve straight mesh ?

Given

A piecewise linear mesh + \mathcal{NURBS} curves on the boundary

Idea

Using linear analogy on control points subdivided mesh.

Thanks to convex hull property,

Subdivided mesh legal \Rightarrow NURBS mesh valid.

Steps:

- create mid-edge points and consider them as control points,
- subdivide the mesh with those points,
- deform the subdivided mesh using elasticity analogy to fit with \mathcal{NURBS} boundary curves.

How to curve straight mesh ?



Linear elasticity analogy

Solving on the initial control points mesh \mathcal{T}_h^0 :

div
$$\left(\lambda \operatorname{tra}(\nabla u) + \mu(\nabla u + \nabla u^T)\right) = 0 \text{ on } \Omega$$

 $u = g \text{ on } \partial\Omega$

with $\lambda > 0$ and $\mu > 0$.

The deformed control points mesh \mathcal{T}_h^D :

$$M^D = M^0 + u(M^0)$$

with M^D vertex of \mathcal{T}^D_h and M^0 vertex of \mathcal{T}^0_h

 \Rightarrow Dirichet BD : $g(M^0) := M^D - M^0$.

Linear elasticity analogy

Is the deformed mesh legal ?

No if the boundary deformation is too large. To overcome this problem, we notice that the previous problem is linear.

• Let an element $K^0 = (M^0_1, \dots, M^0_{d+1})$ of \mathcal{T}^0_h and

$$K^D = (M_1^0 + u(M_1^0), \dots, M_{d+1}^0 + u(M_{d+1}^0))$$

the same element in \mathcal{T}_h^D

• A mesh is legal if, for any element K^0 ,

 $vol(K^0)$ and $vol(K^D)$ have the same sign.

Linear elasticity analogy

• Considering

$$\omega_{K_0}: \theta \mapsto \operatorname{vol}\left((M_1^0 + \theta u(M_1^0), \dots, M_{d+1}^0 + \theta u(M_{d+1}^0)) \right).$$

• Find the smallest θ such as there exists one K^0 for which $\omega_{K_0}(\theta) = 0$ is enough to know if resulting mesh is valid.

This amounts to solving a quadratic (in 2D) or a cubic (in 3D) polynomial on all the simplices of T⁰ and look for the smallest root.

Linear elasticity analogy algorithm

- **(**) Solve linear elasticity equations for the BC g,
- 2 Look for the smallest θ , say θ_0 such that for any $\theta \in [0, \theta_0[, \omega_{K^0}(\theta) > 0 \text{ for any } K^0 \text{ in } \mathcal{T}^0$, then
 - if $\theta_0 \geq 1$, the final mesh is obtained,
 - if $\theta_0 < 1$, then
 - update the initial mesh with

$$M^D = M^0 + u(\theta M^0),$$

• go os step 1 with $\mathcal{T}^0 = \mathcal{T}^D$ and $g = (1 - \theta_0)g$

Curved mesh examples



Isogeometric numerical simulations

Euler or Navier-Stokes equations

Problem to solve: $\begin{cases} \overrightarrow{\nabla}.\overrightarrow{\mathcal{F}}(U,\overrightarrow{\nabla}(U)) = 0, \quad \forall x \in \Omega\\ \text{Boundary Conditions } (\mathcal{BC}) + \text{Initial Conditions} \end{cases}$

Isogeometric numerical method:

• Residual distributed scheme : generalization of SUPG finite element (Abgrall and al)

Differences between isogeometrical analysis and classical one:

- Derivates of basis functions not constant on an element ۲
- Degrees of freedom not necessary interpolated

Isogeometric numerical results



(e) Euler subsonic



(f) Euler supersonic



(g) Navier Stokes subsonic



(h) 3d test case

Subsonic flow around 2 cylinders: Ma = 0.38



Entropy error for P2 simulation

Subsonic flow around 2 cylinders: Ma = 0.38



Entropy error for NURBS simulation

Subsonic flow around a cylinder: Ma = 0.38

 17×64 vertices mesh



Subsonic flow around a cylinder: Ma = 0.38

Entropy error: P2 simulation vs NURBS simulation



Maximum error reduced by 76%

Mesh



- 240 vertices
- order 3 : 840 degrees of freedom
- order 4 : 1969 degrees of freedom



(j) Classic scheme of order 3



(l) Isogeometric scheme of order 4

Mesh refinement



(n) Isogeometric scheme of order 3

Subsonic flow over a bump: Ma = 0.5Entropy error







(p) Isogeometric scheme

Supersonic flow over a NACA0012: Ma = 1.2

Mesh refinement



Supersonic flow over a NACA0012: Ma = 1.2

Mach number



P2 turbulent NS simulation ONERA M6 wing

- inflow mach number: $M_{\infty} = 0.8395$,
- Angle of Attack: $\alpha = 3.06^{\circ}$,
- Reynolds number: $Re = 11.72 \times 10^6$



Figure: meshes for Dassault wing test case.

P2 turbulent NS simulation



(a) C_p coefficient.



(b) Cross section of the pressure coefficient at $44\%,\,65\%$ of the chord compared to experimental results.

Conclusion

Done work

- Génération automatique de maillage Bézier/NURBS meshes.
- Génération de maillages courbes avec couches limites.
- Simulation Euler and Navier-Stokes avec schémas aux rédidus en formulation P2 Lagrange et isogéométrique.

Perspectives

- Traitement des frontières courbes,
- Modification de maillages courbes (adaptation),
- Ordre très élevé.